

Lebensdauerabschätzung von Faserverbundbauteilen

DI Martin Fritz, DI Peter Reithofer, DI Pichler Michael

Die auftretenden Versagensmechanismen und die zugeordneten ertragbaren Lastspiele werden bei Faserverbundbauteilen von zahlreichen Faktoren beeinflusst.

Lokale Faserorientierung und Faseranteil, Umgebungseinflüsse wie Temperatur und Feuchte, chemische und physikalische Alterungsvorgänge, mögliche Energiedissipationen oder herstellungsbedingte Eigenspannungen können die Lebensdauer maßgeblich verkürzen.

Umso wichtiger ist daher, die Berücksichtigung der wesentlichen Einflussfaktoren in der Prozesskette und die Verwendung hinreichend geeigneter Versagenshypothesen bei der virtuellen Lebensdauerabschätzung von Faserverbundbauteilen.

Anhand von Anwendungsbeispielen aus der Automobilindustrie wird die Vorgehensweise erläutert und Effekte werden diskutiert.

Fatigue of composite parts

The occurring failure modes and their dedicated tolerated number of load cycles of composite parts are influenced by many parameters.

Parameters like fiber-orientation und fiber-content, environmental conditions such as temperature and humidity, chemical and physical aging, dissipation of energy and residual stresses emerging during the production process have a wide impact on the fatigue life of composite parts.

Therefore it is very important to consider the main influencing factors and to find an adequate failure criterion for the virtual fatigue-prediction of composite parts.

The principal method and the occurring effects are shown on the basis of examples from the automotive industry.

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Lebensdauerabschätzung von Faserverbundbauteilen (fatigue of composite parts)

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group of companies



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I N P H Y S I C S W E T R U S T

product development
lightweight applications / polymers and composites



...is this predictable?

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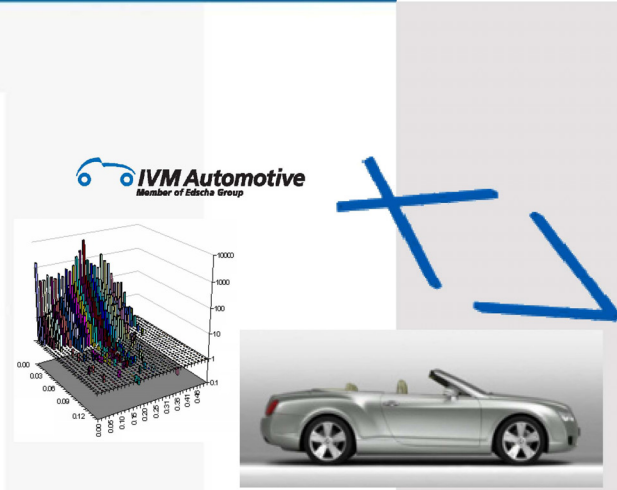
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product development
lightweight applications / polymers and composites



LH₂ – inner tank suspension
lowest possible heat transfer / BMW clean energy
high stiffness, high strength composite solution
increased performance : 250%



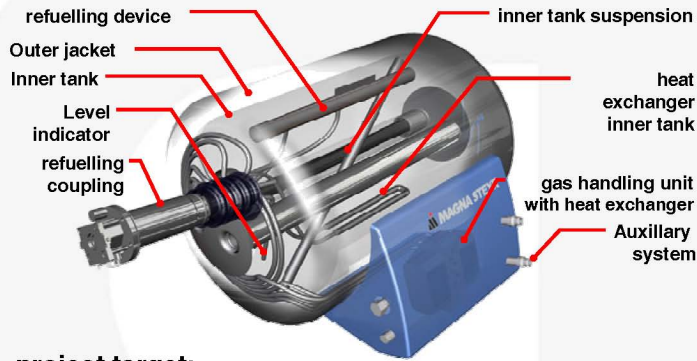
4a fatigue - composites
linear cumulative damage analysis
failure prediction by Puck's criteria
consideration of anisotropic lay-up

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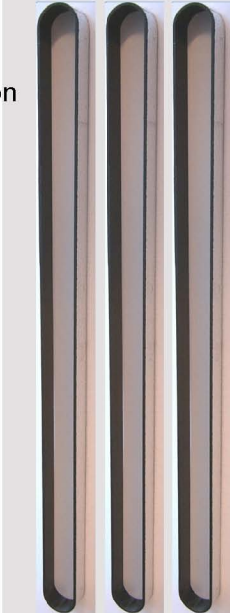
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example: development LH2 - tank suspension



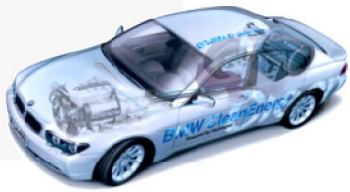
reasons for using composites:

- stiffness
- strength
- Weight
- heat conduction



project target:

- as low as possible warmth penetrate into the inner tank
- max. place exploitation – filling volume
- observance of the load requirement during the drive and the crash



project: Kryogener LH2 - Tank
partner:

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LH2 - tank suspension defining test setups

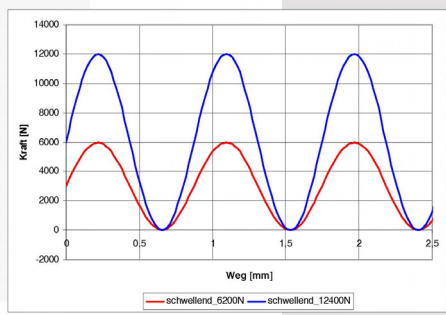
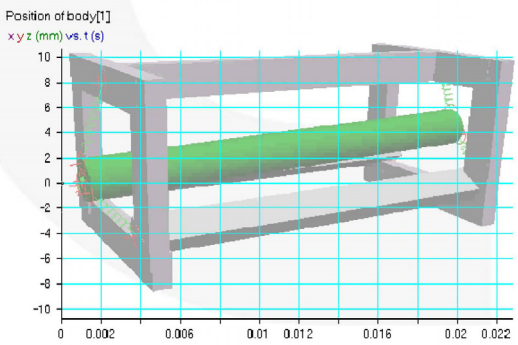


tests on bad gravel roads



load kN	speciments	frequency Hz	bolt angle	load cycles ca.
test1: part load/cycle curve at 77K				
6.5	3	10	3	1000
9.1	3	10	3	10000
10.1	3	10	3	50000
12.1	3	10	3	1000000
test2: reference with paralell bolts at 77K				
14.5	3	10	0	50000
test3: reference at room temperature				
14.5	3	10	0	50000
14.5	3	10	3	50000

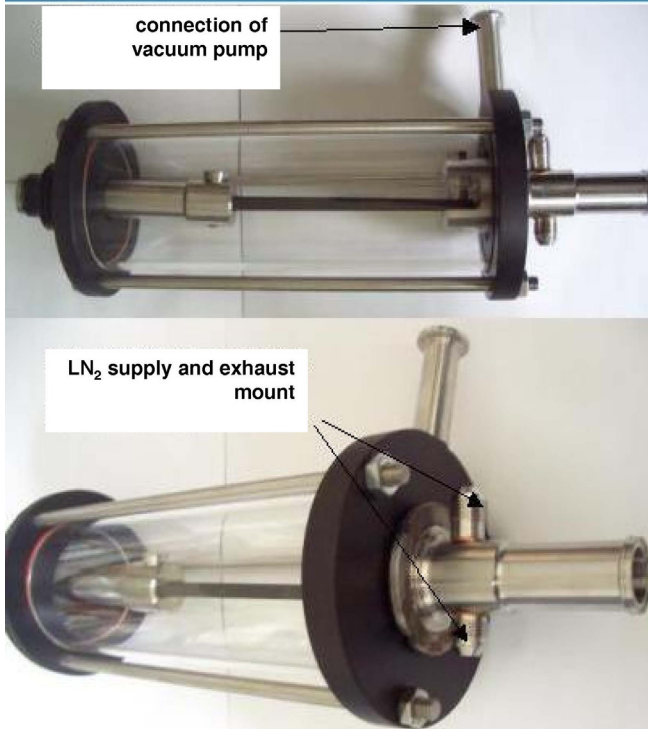
multi body simulations



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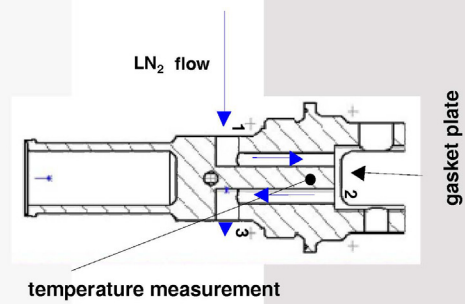
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LH2 - tank suspension equipment and test setup



realistic test setup

- One side is cooled to 77K
- High vacuum inside
- Angular position of Bolts
- Additional isolation during test – radiation shields

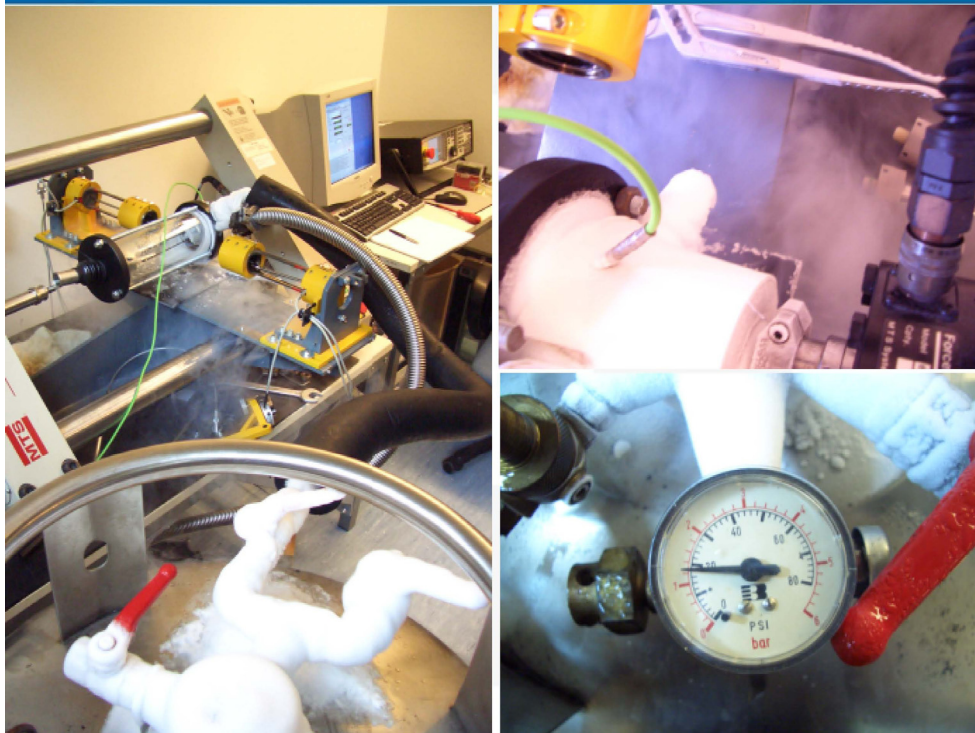


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LH2 - tank suspension equipment and test setup



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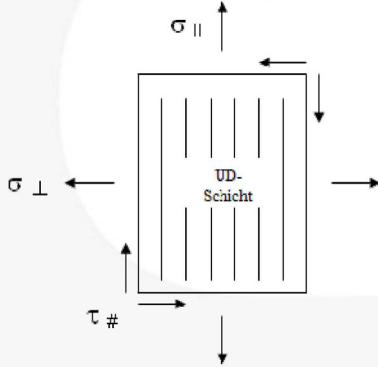
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LH2 - tank suspension CFRP – unidirectional Layer

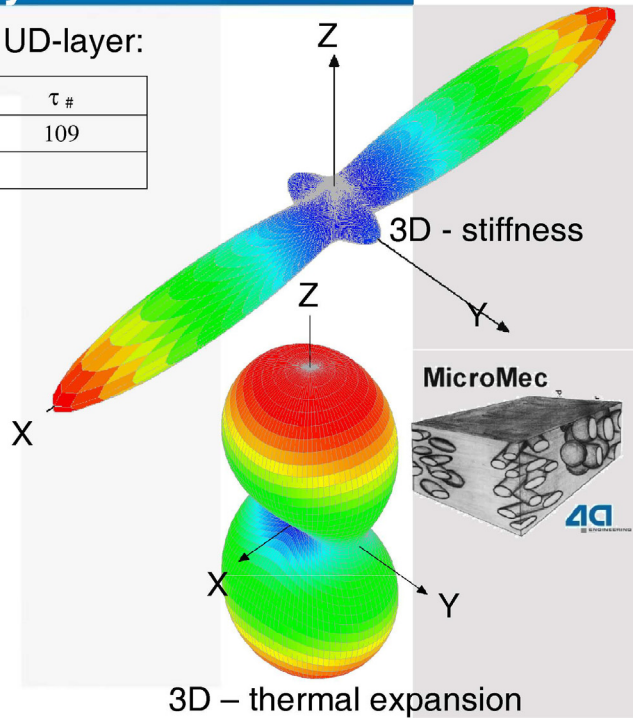


Typical strenght values of a CRFP - UD-layer:

IM7/977	$\sigma_{ }$	σ_{\perp}	$\tau_{\#}$
Zug	1430	36	109
Druck	-900	-218	



- 2-D ply stress
- local coordinating system
- $\tau_{\#}$ interlaminar shearstrenght



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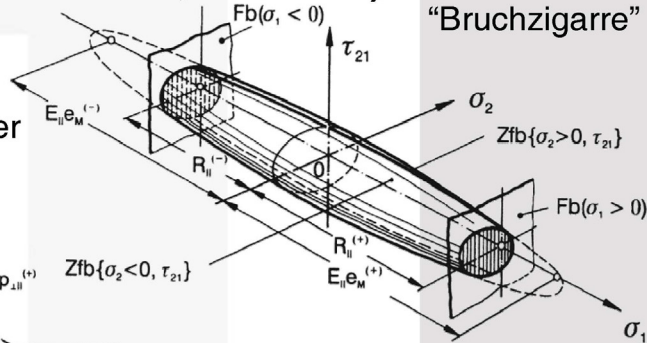
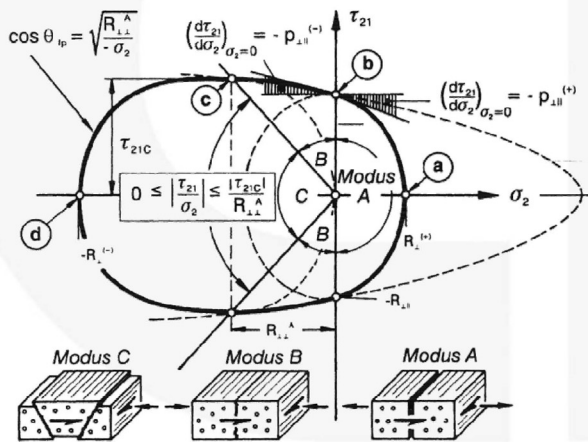
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LH2 - tank suspension puck's law



combined failure criteria (puck's law, hashin...)

- fiber- and matrix-failure mode
- standby factor for matrix und fiber



- matrix-failure mode :
tension
shear
"Sprengbruch"
- also for 3D stress states are covered

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LH2 - tank suspension modeling the fatigue-strength

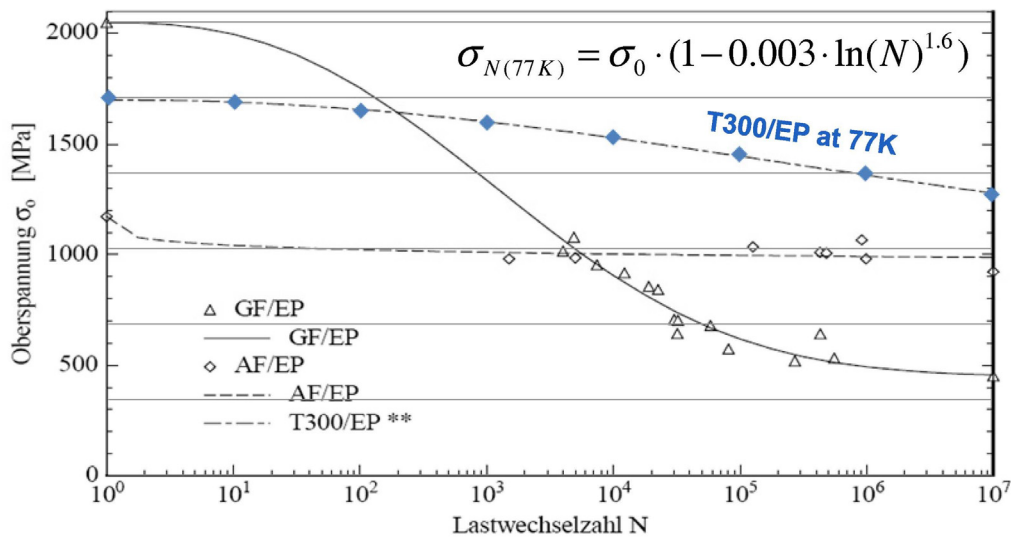


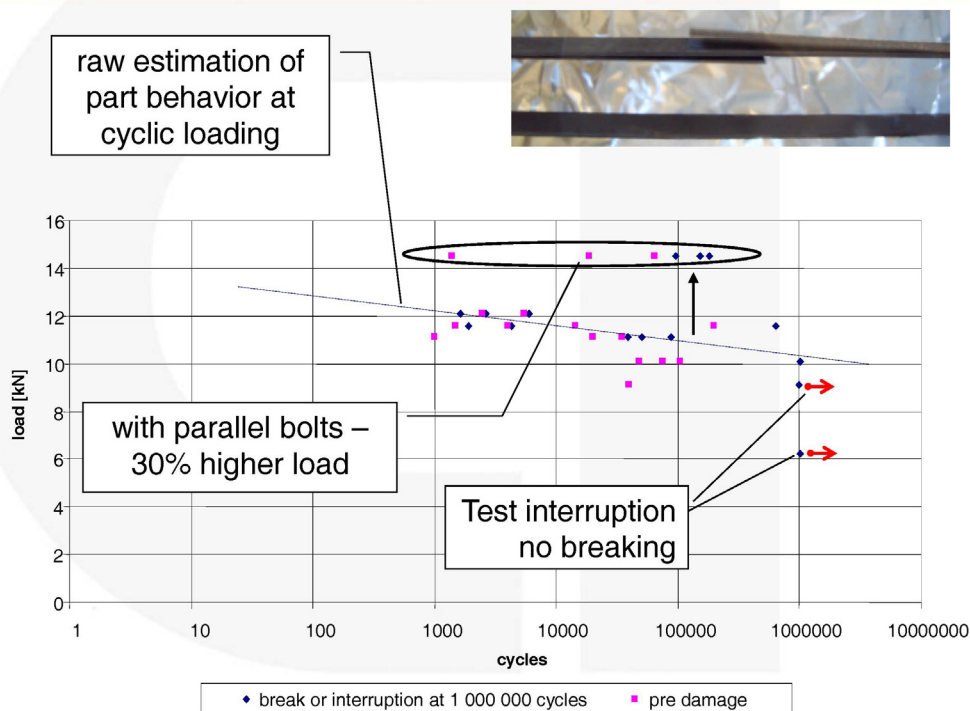
Abbildung 5.8: σ_0 -N Diagramm verschiedener UD Verbunde unter schwelliger Zugbelastung bei 77 K; R=0,1
 **) aus [Pannkoke 92], S. 68

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LH2 - tank suspension cyclic loading at 77K

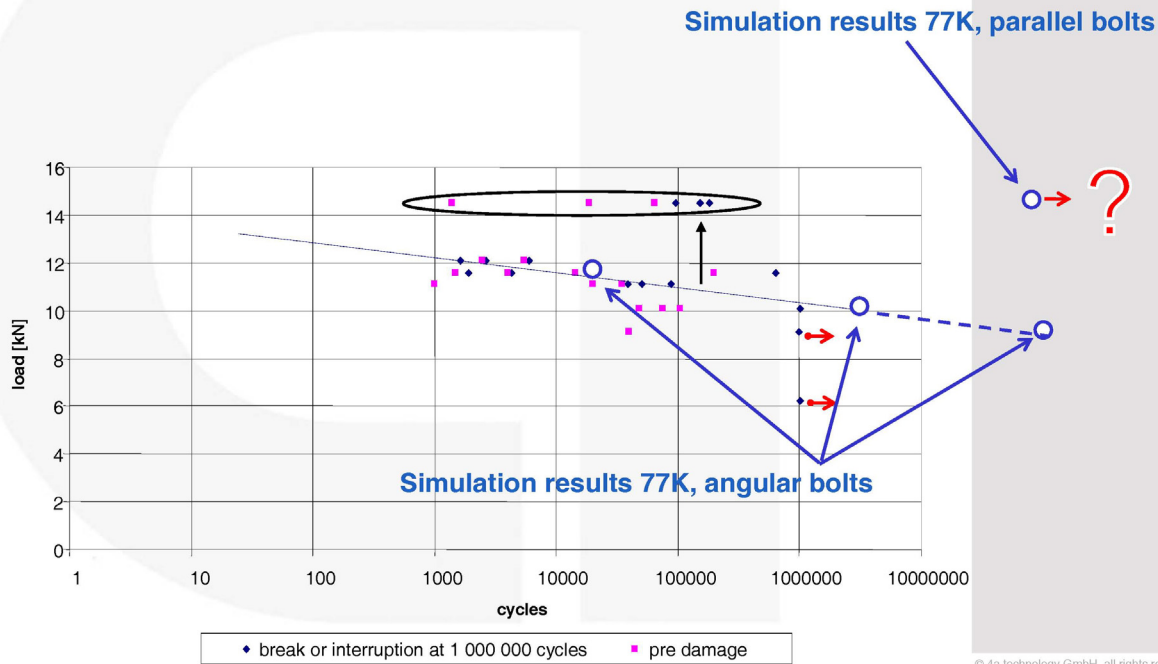


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LH2 - tank suspension prediction of load cycles

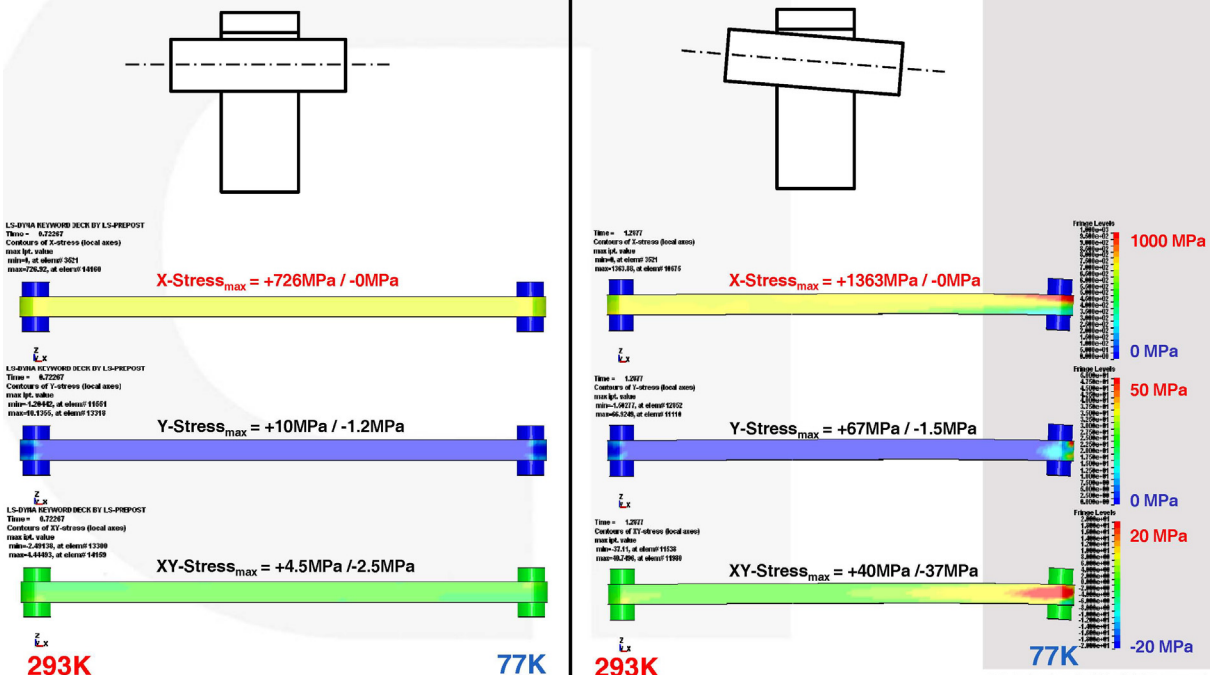


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LH2 - tank suspension simulation - parallel bolt position (at 10kN)



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LH2 - tank suspension

S-N-curves, effect of temperature

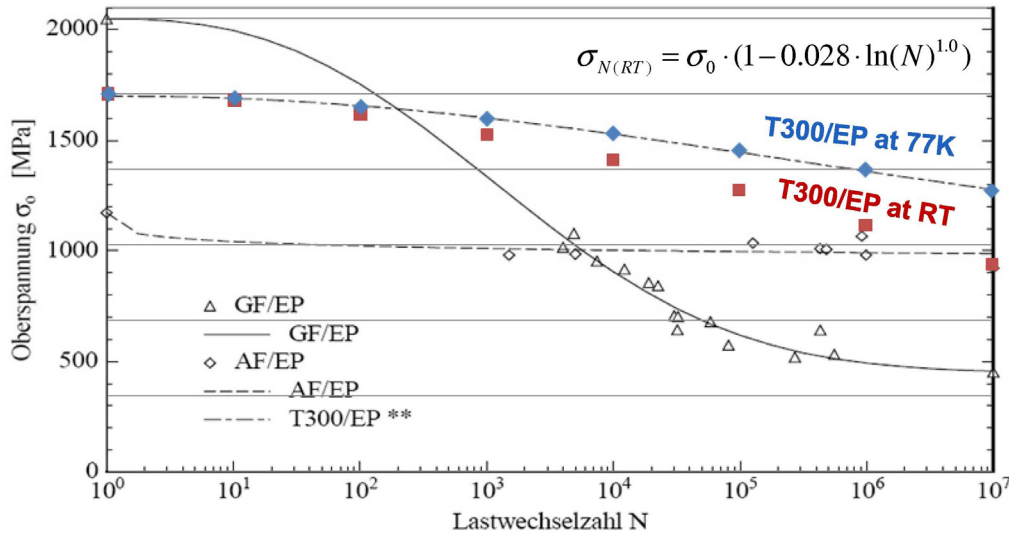


Abbildung 5.8: σ_0 -N Diagramm verschiedener UD Verbunde unter schwellonder Zugbelastung bei 77 K; R=0,1
 **) aus [Pannkoke 92], S. 68

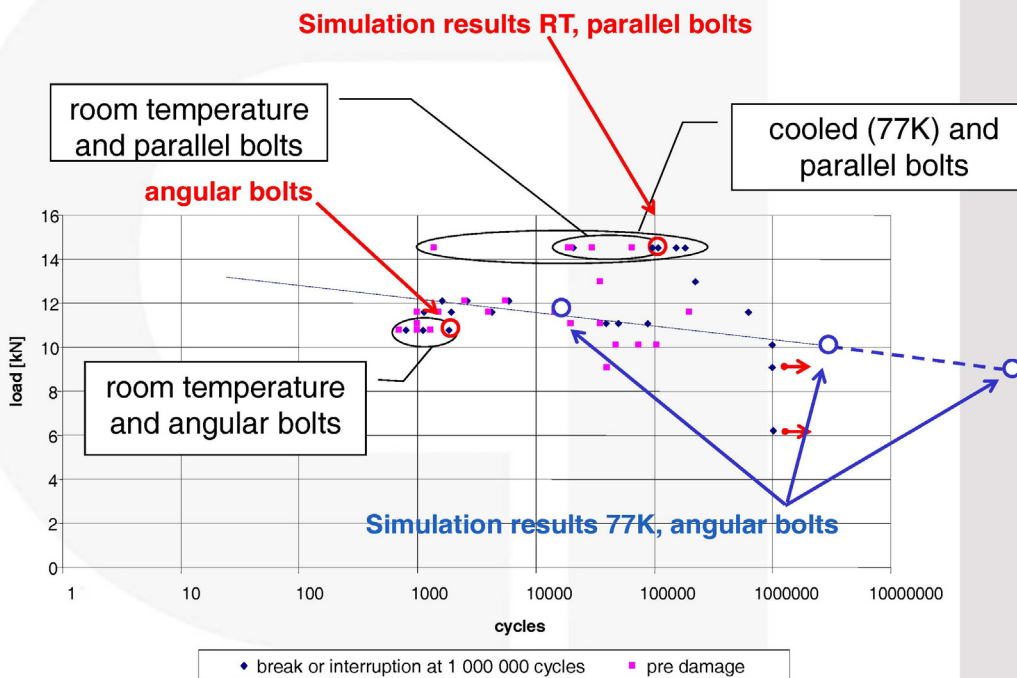
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LH2 - tank suspension

comparison with simulation results

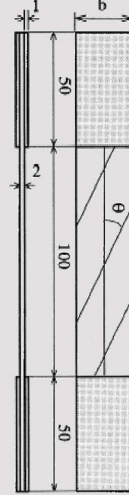
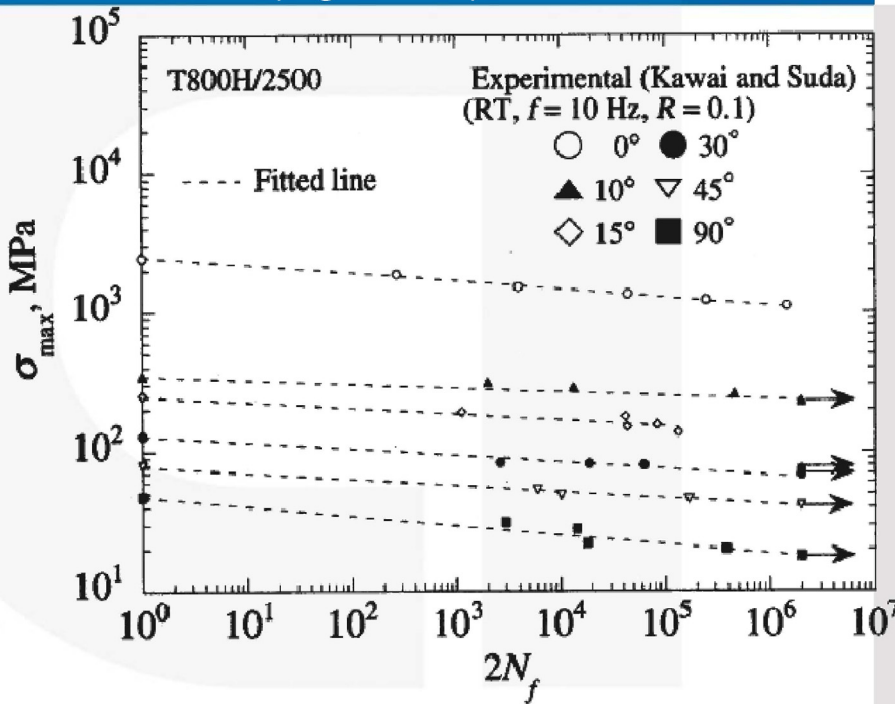


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LH2 - tank suspension effect of stress ratio (e.g. CFRP)

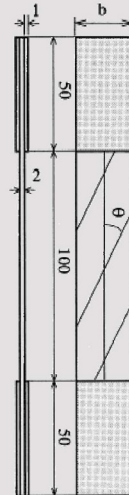
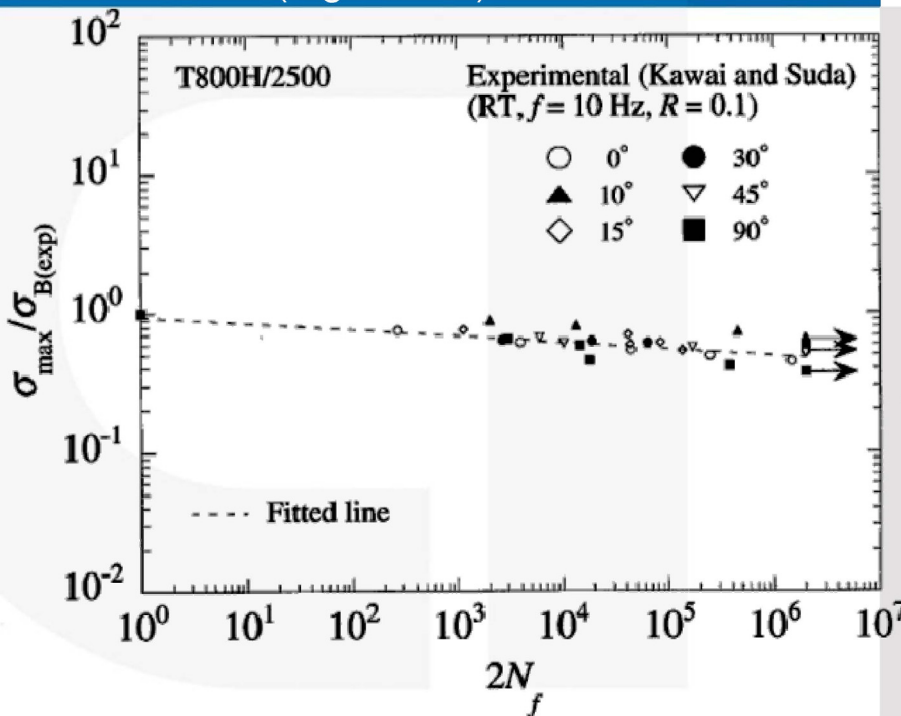


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I N P H Y S I C S W E T R U S T

LH2 - tank suspension effect of stress ratio (e.g. CFRP)

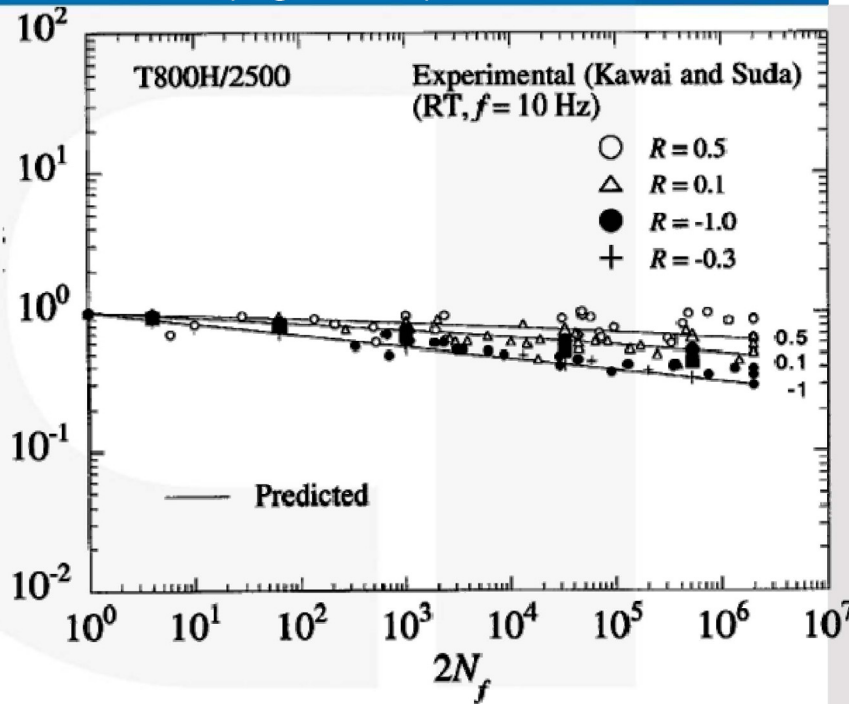


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LH2 - tank suspension effect of stress ratio (e.g. CFRP)

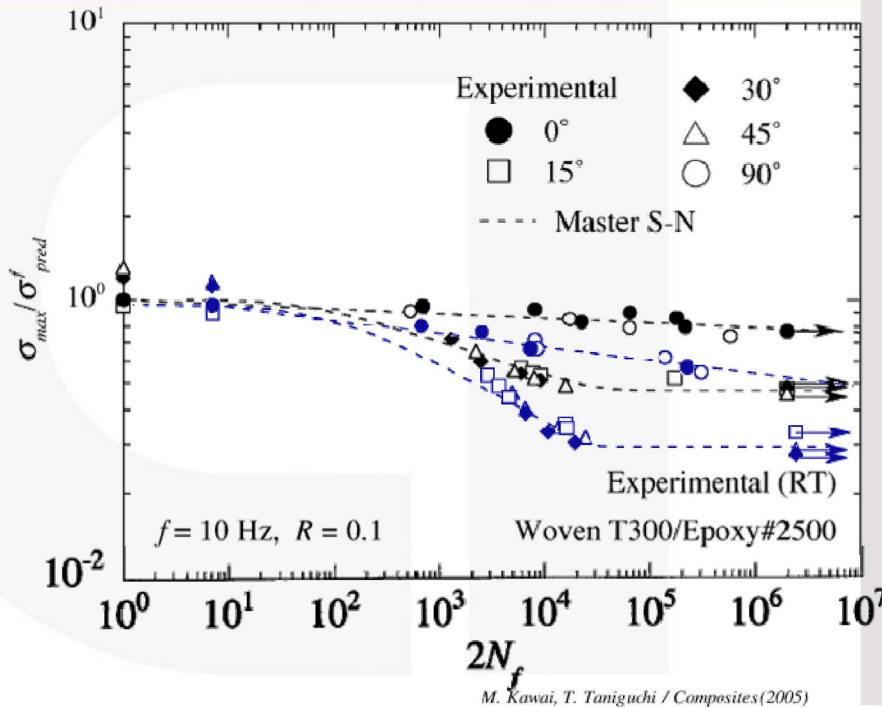


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LH2 - tank suspension effect of off- axis load on fatigue behaviour



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product development
lightweight applications / polymers and composites



LH₂ – inner tank suspension
lowest possible heat transfer / BMW clean energy
high stiffness, high strength composite solution
increased performance : 250%

4a fatigue - composites
linear cumulative damage analysis
failure prediction by Puck's criteria
consideration of anisotropic lay-up

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LH2 - tank suspension
realistic loads – e.g. bad gravel road



Strain versus time

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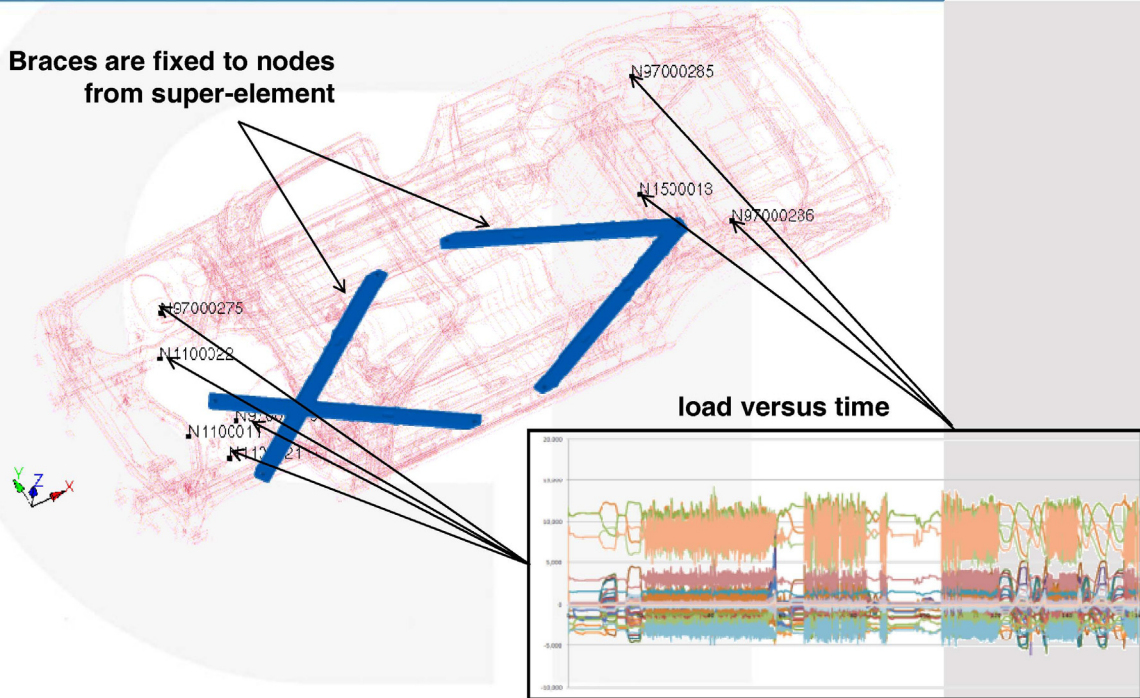
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VX - Brace

complete FE-model (super-element)



Braces are fixed to nodes from super-element



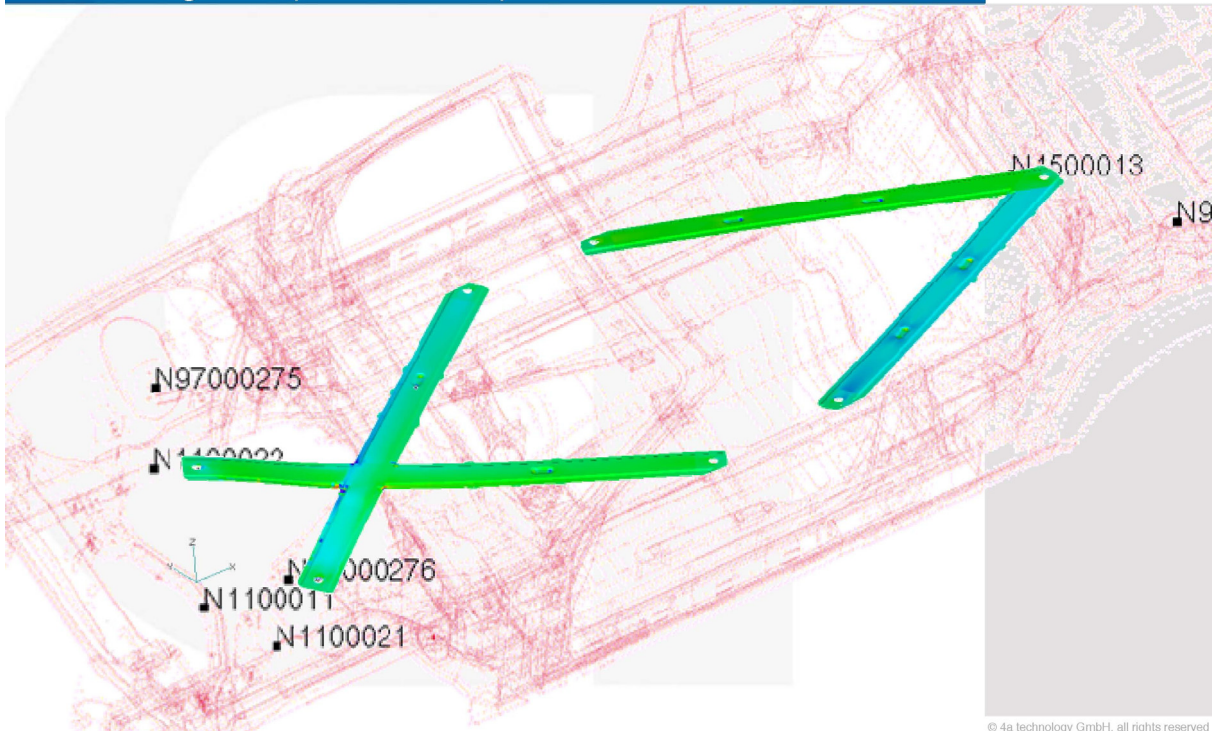
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VX - Brace

critical regions (fiber failure)

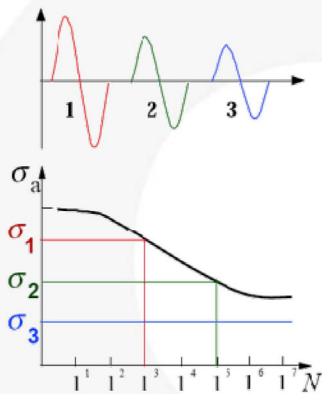


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VX - Brace damage accumulation



$$\sum_{i=1}^I \frac{n_i}{N_i} = \frac{500}{10^3} + \frac{10^3}{10^5} + \frac{10^4}{\infty} = 0.51 < 1$$

◊ Assume that, during the service life, we have 500 loadings of type 1 (defined by mid-value and magnitude), 1000 loadings of type 2 and 10000 loadings of type 3

◊ The Palmgren - Miner rule states that failure occurs when

$$\sum_{i=1}^I \frac{n_i}{N_i} = 1$$

where n_i is the number of applied load cycles of type i , and N_i is the pertinent fatigue life

damage accumulation

$$D = \sum \frac{n_i}{N_i}$$

Original-Miner

$$S_a > S_{aD} : N = N_D \cdot \left(\frac{S_a}{S_{aD}}\right)^{-k}$$

elementare Miner-Rule by Palmgren

$$S_a \leq S_{aD} : N = N_D \cdot \left(\frac{S_a}{S_{aD}}\right)^{-k}$$

Miner-Rule modified by Haibach

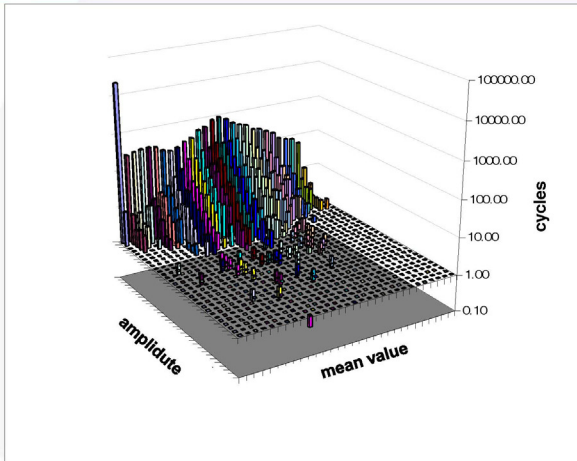
$$S_a \leq S_{aD} : N = N_D \cdot \left(\frac{S_a}{S_{aD}}\right)^{-(2k-1)} \quad S_a > S_{aD} : N = N_D \cdot \left(\frac{S_a}{S_{aD}}\right)^{-k}$$

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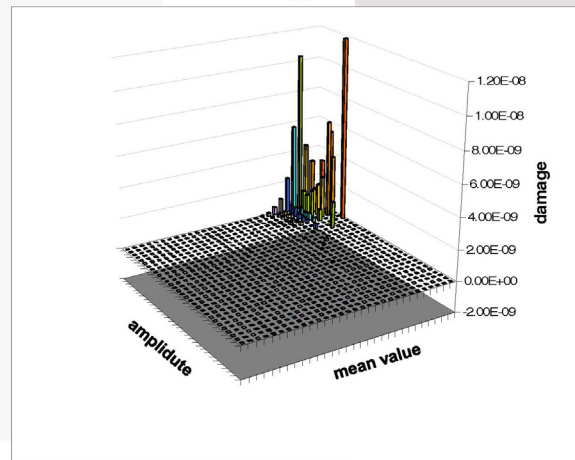
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VX - Brace a critical Element - loadcase R1



total damage, classified by rainflow analysis in mean values and amplitudes of damage (Puck's law)



Number of cycles, classified by rainflow analysis in mean values and amplitudes of damage (Puck's law)

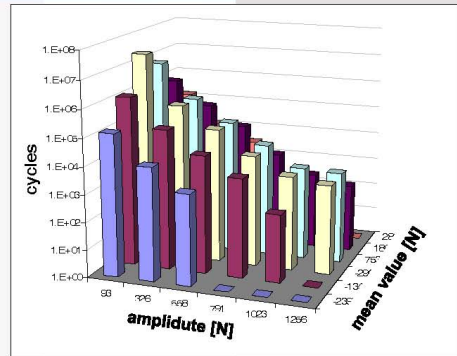
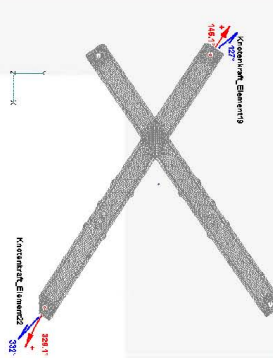
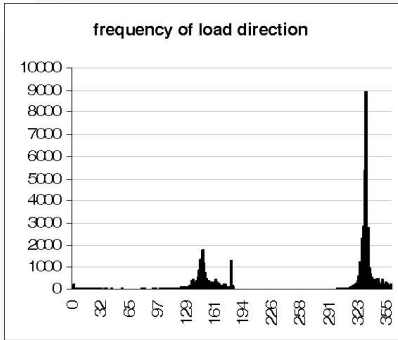
Total damage after 575 rounds → 3.75E-01

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VX - Brace parameters for component tests

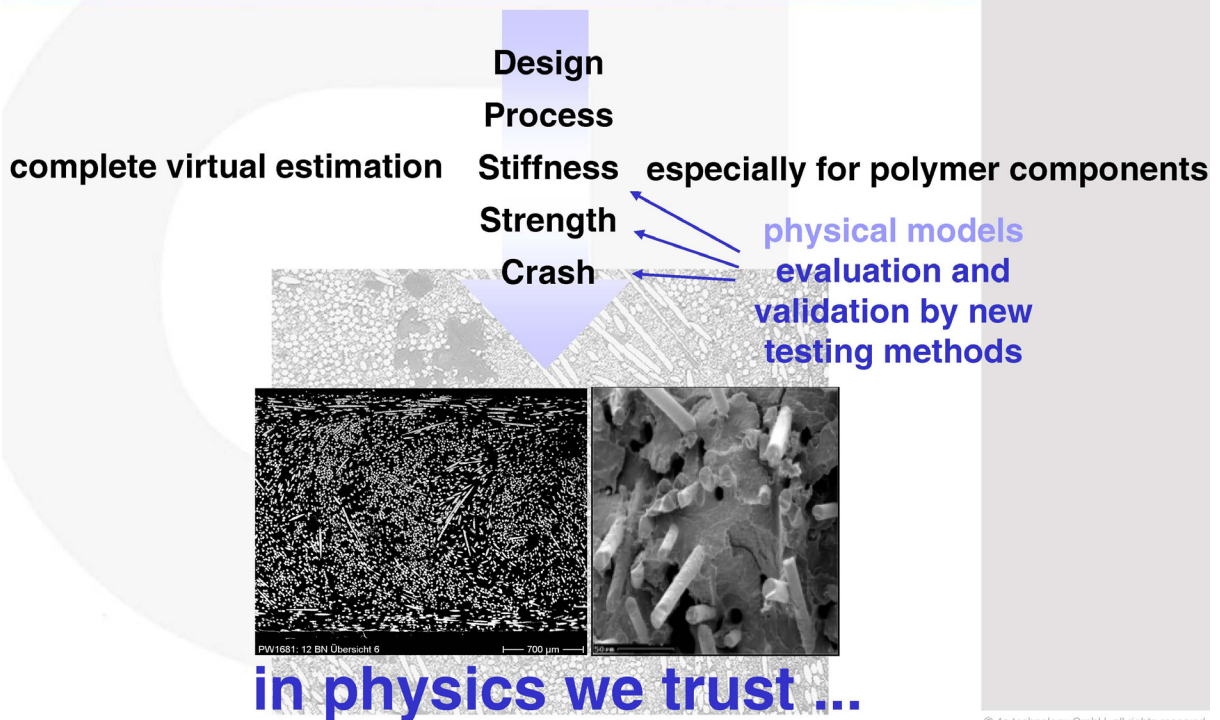


cracks short before end of test (n=87654, +/-10kN)

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vision - virtual estimation from process to break



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