



Gestamp BIW



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Cross Car Beam Multi Optimization

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Summary

1. Introduction

- I. Gestamp Corporation
- II. Gestamp Automotive Products Portfolio

2. The project

- I. Context
- II. Load cases

3. The optimization

- I. Optimization Flow-Chart
- II. First Optimization Phase (NVH)
- III. Second Optimization Phase (NVH and Crash)

4. Results and conclusions

- I. Optimization results
- II. Conclusions







Gestamp Corporation



- Body-in-White, Chassis and Mechanisms
- Wide range of technologies.

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3

Leadership



II. Gestamp Automotive Portfolio





2. The Project

I. Context

Cross Car Beam Assembly



Cross Car Beam is a support that usually holds all Instrument Panel including HVAC System, Knee Airbags, **Steering Column**, Radio and many other components.

Previous Optimization Experience

- Simple morphing shapes (beads, flanges)
- Material properties
- Modal and static analysis

New challenges of this New Project

- Complex morphing shapes including remeshing
- · Components position displacement
- Welding projection after shape modification
- Include both static and dynamic analysis

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2. The Project



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6



I. Optimization Flow-Chart



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7



I. Optimization Flow-Chart

Morphing process





II. First Optimization Phase (NVH) – DOE studies

Morphing examples







II. First Optimization Phase (NVH) – DOE studies

Design variables studied

Example driver's tube morphing



Shape X Right



Shape Z Right



Shape 2 Kight

Shape X Left







Correlation matrix



VARIABLES			
Side bracket position	Passenger's tube shape Z Right		
Passenger's tube position X	Driver's tube shape X Left		
Passenger's tube position Z	Driver's tube shape Z Left		
Driver's tube position X	Driver's tube shape X Right		
Driver's tube position Z	Driver's tube shape Z Right		
Column position X	Driver's tube shape X Middle		
Passenger's tube size	Driver's tube shape Z Middle		
Passenger's tube size Left	Passenger's tube shape X		
Passenger's tube size Right	Passenger's tube shape Z		
Driver's tube size	Driver's tube shape X		
Driver's tube size Left	Driver's tube shape Z		
Driver's tube size Right	Thickness properties		
Passenger's tube shape X Left	Materials properties		
Passenger's tube shape Z Left			
Passenger's tube shape X Right			







II. First Optimization Phase (NVH) – DOE studies

Analysis Results - Metamodels



Steering wheel stiffness

Deflection test

Metamodels were used to redefine the range of the design variables.

For example the driver's tube diameter range changed from [40-70]mm to [55-70]mm

First Phase Results

- After this first optimization phase the amount of design variables was reduced from 35 to 10
- 2. The design space was also reduced changing the design variables range to the place of the best response results.



11

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III. Second Optimization Phase (NVH and Crash)

Design Variables

VARIABLES ADDED	
Firewall shape top	
Firewall shape bottom	
Firewall shape side	
Firewall shape width	
Passenger's tube material	
Driver's tube material	
Tunnel's leg material	

Some design variables were added for frontal crash optimization.

In crash analysis materials stress strain curves were used as design variables as well.





12



III. Second Optimization Phase (NVH and Crash)

DOE Study

DOE study was used to analyse and have more information about the new variables.



History of section force on driver's tube - Side impact

Variables and responses after DOE

VARIABLES	RANGE
Passenger's tube position X	[0 – 50] mm
Passenger's tube size	[25 – 40] mm
Driver's tube position X	[0 – 60] mm
Driver's tube size	[40 – 50] mm
Firewall bracket shape top	[0 – 7] mm
Firewall bracket shape bottom	[0 – 7] mm
Firewall bracket shape side	[0 – 7] mm
Firewall bracket shape width	[-10 – 15] mm
Passenger's tube thickness	[1, 1.5] mm
Driver's tube thickness	[1.5, 2.0, 2.4] mm
Tunnel leg thickness	[1.0, 1.5] mm
Passenger's tube material	[1, 2]
Driver's tube material	[1, 2]
Tunnel leg material	[1, 2]

RESPONSES	TYPE	SIMULATION
1st frequency	Constraint	Modal analysis
2nd frequency	Constraint	Modal analysis
Weight	Objective	-
Deflection	Constraint	Deflection test
Steering wheel displ.1	Constraint	Steering stiffness
Steering wheel intrusion	Constraint	Frontal crash
Reaction force	Constraint	Side impact

Design variables and its ranges after DOE studies





III. Second Optimization Phase (NVH and Crash)

Calculation time distribution (minutes)



- Modal analysis
- Deflection test
- Steering wheel stiffness
- Frontal crash
- Side impact

Optimization used

In order to achieve the best variables configuration and reduce the number of experiments it's used Sequential Response Surface Method (SRSM).



- LS-OPT configuration:
 - Metamodel:
 - Polynomial Quadratic
 - Point selection:
 - Space Filling default 181 points

Optimization history (weight)



The optimization calculated about 20 iterations and more than 220 different configurations. Total optimization time took 3 days (4 CPU's).





4. Results and conclusions

I. Optimization results (NVH and Crash)

NVH results





4. Results and conclusions

I. Optimization results (NVH and Crash)

Crash results



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4. Results and conclusions

II. Conclusions

In the beginning of this project the packaging was very restrictive, the position of the components could not move too much as in a **RFQ phase** and the final optimization **did not achieved very good results**.

Therefore we decided to ignore some parts of the packaging as it were a **concept phase**. Increasing the movement of the components we could check for possible positions which we had been never tried before and see they could be a good solution design.

The final weight reduction is about 18%. (5,70 kg to 4,67 kg) achieving all targets:

- · Modal analysis,
- Deflection test
- Steering column stiffness
- Frontal crash
- Side impact

LS-OPT is a useful and great tool to coordinate different kind of simulations and analyse the results.

However, working with large number of variables could carry some difficulties to manage all together.

In this project a great part of the time was dedicated in the FE-Model parameterization, welding scripting and learning LS-Opt features which we hope to reduce this time for next projects.





