Current Status of Subcycling and Multiscale Simulations in LS-DYNA

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- Explicit time integration using different time steps in different subdomains
- How is one to treat boundary between domains?
 - Spurious wave propagation across domains
 - Different proposals
- Does it really work?
 - Some say yes, some say no
 - The implementation is undertaken with a positive mindset it will work !!!





Finite element mesh in 2D

- Material stiffness E
- Material density ρ
- Characteristic length l and thickness t





Analyze the behavior of a particular node
Surroundings act as boundary conditions





Inertial mass internally represented by nodal masses
m = ρl²t









Frequency and stable time step of node

$$\omega = \sqrt{k/m} = \frac{1}{l}\sqrt{E/\rho}$$
$$\Delta t = l\sqrt{\rho/E}$$



Generalization in explicit analysis - no subcycling



Global time step is smallest element time step

$$\Delta t = \min_{i=1,\dots,N} \Delta t_i = \min_{i=1,\dots,N} l_i \sqrt{\rho_i/E_i}$$



Locality in subcycling



Assume each node can have individual time steps

$$\Delta t^{j} = \min_{i \in M_{j}} \Delta t_{i}$$
$$\Delta t^{10} = \min(\Delta t_{4}, \Delta t_{5}, \Delta t_{7}, \Delta t_{8})$$



Synchronization restriction - nodal simulation time step



Each time step must be a 2-power multiple of the smallest step

- $\Delta \tilde{t}^{j} = \max_{\Delta \tilde{t}^{j} < \Delta t^{j}} (\Delta t, 2\Delta t, 4\Delta t, 8\Delta t, 16\Delta t, 32\Delta t, 64\Delta t)$
 - $\Delta \tilde{t}^{10} = 4\Delta t$

The subcycling nodal integration scheme complete !!!



Element simulation time step - internal forces



Element time step must not exceed surrounding nodes

 $\Delta \tilde{t}_i = \min_{j \in N^i} \Delta \tilde{t}^j \qquad \Delta \tilde{t}_8 = 2\Delta t$

Nodal coordinates act as boundary conditions for element calculations
The constant velocity approach - nodal coordinates linearly interpolated
Subcycling algorithm complete !!!



The global synchronization time step



Large element simulation time steps save CPU time
CPU time gain is limited by the partitioning of elements/nodes into time step groups

Largest element time step is called synchronization step

$$\Delta T = \max_{i=1,\dots,M} \Delta \tilde{t}_i \qquad \Delta T = 8\Delta t$$

Specifying the synchronization step - keyword input



*CONTROL_SUBCYCLE_{N}

- N=2,4,8,16,32,64
- *CONTROL_SUBCYCLE_4

N used for conservative or aggressive approach, default 16
Motivated by the lack of theoretical results



Time step restrictions - external forces

- Elements or nodes with the following properties are executed with the smallest time step
 - Rigid materials
 - Connections with rigid elements, including nodal rigid bodies
 - Connections in other ways than just by elements are treated the same way
- Time step ordering between elements is maintained by adding sufficient amount of mass, for efficiency
 - Mass scaling is ALWAYS on for subcycling
- All other features, not related to elements, are expected to run stable at some independent time step
 - Contacts in particular
 - *CONTROL_SUBCYCLE_{N}_{M}
 - M specifies the maximum time step for external forces
 - M=1,2,4,8,16,32 or 64, default 1



Multiscale option when using mass scaling

If mass scaling is invoked, DT2MS<0, then this works similar to "normal" mass scaling</p>

 $\Delta \tilde{t}^{j} = \max(\Delta t_{ms}, \Delta \tilde{t}^{j}) \qquad \Delta \tilde{t}_{i} = \max(\Delta t_{ms}, \Delta \tilde{t}_{i})$

Assume that a target time step is desired

Reached by applying mass scaling

Highly refined parts will then be given too much inertia for its own good

We want to specify a smaller mass scaled time step for these

Use *CONTROL_SUBCYCLE_MASS_SCALED_PART_{SET}

- Allows for decreasing or increasing time step for individual parts
- Specify part and associated time step
- For this option, all part time steps are set by the user
- Algorithm stays the same
 - Multiscale option



S-rail - refined spotwelds



- S-rail problem run with and without subcycling
 - 24640 shells, 55296 solids
 - 79 % of the nodes run on smallest time step
- Check spotweld contact and refined solid spotweld Used with *CONTROL_SUBCYCLE_64_4
 - No subcycling 3757 seconds
 - Subcycling 3406 seconds (9% CPU savings)
 - Multiscale 3 levels, 3454 seconds (8 % CPU savings)



S-rail - results



Results invariant to subcycling option

Timings are intuitive

	Subcycling, N=64, M=4	Multiscale, M=4	No subcycling
Contacts	78	79	241
Elements	1783	1790	2141



RAV4 - use of multiscale



10 ms side hit simulation, 7 kg cylinder, 30 mp
501587 shells, 21539 solids, B-pillar refined

Multiscale

- DT2MS=-1e-3
- DT(B-pillar)=1.3e-4

Subcycling

- *CONTROL_SUBCYCLE_64_4
 - 7 % of the nodes run at smallest time step



RAV4 - element partitioning



No subcycling

Subcycling 795 s

Multiscale 906 s

1443 s

(45 % CPU

(37 % CPU

savings)

savings)

RAV4 - results

Energies and contact force

Results invariant to subcycling option

Rav4 CPU timings	Subcycling, N=64, M=4	Multiscale, M=4	No subcycling
Contacts	133	133	288
Elements	194	206	636



Camry - robustness of subcycling

2012 TOYOTA CAMRY MODEL (NCAC V01) Time = 0



- 1603439 shells, 64257 solids
- Various subcycling options and contacts
- CONTROL_SUBCYCLE_16_1, SOFT=0 contact, SFS=0.1
- Unfortunately, 91% of the nodes run on smallest time step



Camry - results

#CPU	No subcycling	Subcycling
12	28942	28091
24	14964	14158
48	7578	7642
96	4122	4028



Internal energy and rigid wall force from subcycling and no subcycling



Conclusions and future work

Subcycling implemented and tested in fairly advanced applications

Promising

Some further investigations are in place

- Contacts in particular
- Alternate interface algorithms(?)
- Continuous update of material time step database
- If used properly some saving can be done
 - Inferior to SMS, but stands as an alternative
- Available for usage in R8.0 of LS-DYNA
 - If used, please give feedback



Thank you!



